Program: DTE (2<sup>nd</sup> Sem)

Date: 14 May, 2024 Time: 10:00 am - 01:00 pm (Group A - Morning)

## ISLAMIC UNIVERSITY OF TECHNOLOGY (IUT) ORGANISATION OF ISLAMIC COOPERATION (OIC) DEPARTMENT OF MECHANICAL AND PRODUCTION ENGINEERING

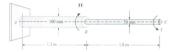
Semester Final Examination	Summer Semester: A.Y. 2022 - 2023	
Course Number: ME 4403	Time	: 03 Hours
Course Title: Mechanics of Materials	Full Marks	: 150

## There are 06 (Six) Questions. Answer all of them.

Each question carries equal marks. Symbols have their usual meanings. Draw the free body diagram if required. The right column also indicates the caurse objective (CO) and Program Outcomes (PO) addressed by each question. Assume reasonable values for missing data.

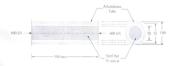
(a) The stepped steel shaft shown in Figure below is subjected to a torque (T) at the free end, and a torque (3T) in the opposite direction at the junction of the two sizes.

What is the total angle of twist at the free end, if maximum shear stress in the shaft is limited to PO2 70 MPa? Assume the modulus of rigidity to be 84 GPa.



(b) A solid steel bar 500 mm long and 50 mm diameter is placed inside an aluminium tube 75 mm 13
inside diameter and 100 mm outside diameter. The aluminium tube is 0.5mm longer than the steel
bar. An axial load of 600 kN is applied to the bar and cylinder through rigid plates as shown in
Figure below.
CO2
PO2

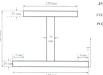
Find the stresses developed in the steel bar and aluminium tube. Assume E for steel as 200 GPa and E for aluminium is 70 GPa



2

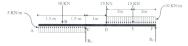
An 1-section girder is used as a simply supported beam over a span of 8 m. The cross section of the girder is shown in Fig. 2(a). And the loading condition is shown in Fig. 2(b). If modulus of elasticitiv is view as E = 100 GPA.

area) of the section



(ii) Draw S.F.D and B.M.D for the Beam

(iii) Use Macaulay's method to find bending moment, slope and deflection at A, C, E (iv) Find M<sub>met</sub> and maximum stress set-up (σ<sub>met</sub>) in the Beam



## 3 (a) Write the difference between column and struts with example

5

COI

PO1

- (b) Draw schematic diagrams of different end conditions in column with corresponding equivalent length 5 equations. PDU
- 4 (a) A material is subjected to two mutually perpendicular direct stresses of 80 MN/m<sup>2</sup> tensile. 30 and 50 MN/m<sup>2</sup> compressive, together with a shear stress of 30 MN/m<sup>2</sup>. The shear couple CO4 acting on planes carrying the 80 MN/m<sup>3</sup> stress is clockwise in effect. Calculate

(i) The magnitude and nature (angle) of the principal stresses.

(ii) The magnitude of the maximum shear stresses in the plane of the given stress system.

(iii) The direction of the planes on which these stresses act.

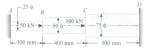
(iv) The magnitude of the normal and shear stress on a plane inclined at 20<sup>e</sup> counterclockwise to the plane on which the 50 MN/m' stress acts.

Confirm your answer by means of a drawing Mohr's stress circle diagram of the problem

5 (a) Derive the following relation of bending moment equation with proper assumptions and schematic diagram.

- $\frac{M}{I} = \frac{\sigma}{y} = \frac{E}{R}$
- (b) Write the relationship equations between deflection, slope, bending moment, shear force and load distribution COI
- 6 (a) A circular steel bar ABCD, rigidly fixed at A and D is subjected to axial loads of 50 kN and 100 kN 12 at B and C as shown in the following figure.

Find the loads shared by each part of the bar and displacements of points B and C. Take E for the steel as 200 GPa.



(b) A block showing in the following figure, weighing 35 kN is supported by three wires. The outer two 8 wires are of steel and have an area of 100 mm<sup>2</sup> acht, whereas the middle wire of aluminum and has con provide the classicity modulus of steel and aluminum are 200 GPa and 80 GPa respectively, then calculate the stresses in the aluminum and steel wires.



## Related Equations:

The stresses on oblique planes owing to a complex stress system are:

normal stress =  $\frac{1}{2}(\sigma_x + \sigma_y) + \frac{1}{2}(\sigma_x - \sigma_y)\cos 2\theta + \tau_{xy}\sin 2\theta$ shear stress =  $\frac{1}{2}(\sigma_x - \sigma_y)\sin 2\theta - \tau_{yy}\cos 2\theta$ 

The principal stresses (i.e. the maximum and minimum direct stresses) are then

$$\begin{split} \sigma_z &= \frac{1}{2}(\sigma_x + \sigma_y) + \frac{1}{2}\sqrt{\left[(\sigma_x - \sigma_y)^2 + 4\tau_{xy}^2\right]} \\ \sigma_z &= \frac{1}{2}(\sigma_x + \sigma_y) - \frac{1}{2}\sqrt{\left[(\sigma_x - \sigma_y)^2 + 4\tau_{xy}^2\right]} \end{split}$$

and these occur on planes at an angle  $\theta$  to the plane on which  $\sigma_x$  acts, given by either

$$\tan 2\theta = \frac{2\tau_{xy}}{(\sigma_x - \sigma_y)}$$
 or  $\tan \theta = \frac{\sigma_p - \sigma_x}{\tau_{xy}}$ 

where  $\sigma_p = \sigma_1$ , or  $\sigma_2$ , the planes being termed principal planes. The principal planes are always at 90° to each other, and the planes of maximum shear are then located at 45° to them.

The maximum shear stress is

$$\tau_{\max} = \frac{1}{2} \sqrt{\left[ (\sigma_x - \sigma_y)^2 + 4\tau_{xy}^2 \right]} = \frac{1}{2} (\sigma_1 - \sigma_2)$$